

Momentum Integral Theorems for the Boundary Layer:

Solution of the boundary layer equations

$$\frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} = -\frac{1}{\rho} \frac{\partial p}{\partial x} + v \frac{\partial^2 u}{\partial y^2} \quad \text{--- (i) (i)}$$

$$\frac{\partial p}{\partial y} = 0 \quad \text{--- 1 (ii)}$$

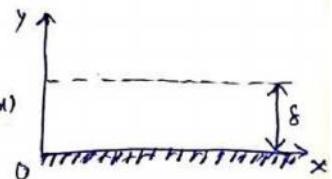
$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0 \quad \text{--- 1 (iii)}$$

is in general very difficult. The sol's obtained previously is ~~a~~ very special case. For engineering problems, it is often acceptable if an approximate solution can be obtained. One of the most useful methods is the Von Karman - Pohlhausen method based on the integral theorems. The basic concept of this method is that the sol's satisfy the diff. eqns only on the average. In other words, it is not anticipated that the sol satisfies the boundary layer differential eqn's at every point (x, y) but the momentum integral eqn and the boundary conditions must be satisfied. The momentum integral eqn is obtained by integrating the boundary layer equations w.r.t y .

- Pohlhausen
The Von Karman / Integral Relation:

Integrating Eqn 1(i) w.r.t y from $y=0$ to $y=\delta(x)$ we have

$$\int_0^{\delta} \frac{\partial u}{\partial t} dy + \int_0^{\delta} u \frac{\partial u}{\partial x} dy + \int_0^{\delta} v \cdot \frac{\partial u}{\partial y} dy = -\frac{1}{\rho} \int_0^{\delta} \frac{\partial p}{\partial x} dy + \frac{1}{\rho} \int_0^{\delta} \frac{\partial^2 u}{\partial y^2} dy$$



$$\int_0^{\delta} \left(\frac{\partial u}{\partial t} \right) dy + \int_0^{\delta} \left(u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} \right) dy = -\frac{1}{P} \int_0^{\delta} \frac{\partial b}{\partial x} dy + \frac{u}{P} \int_0^{\delta} \frac{\partial^2 u}{\partial y^2} dy \quad \text{--- (2)}$$

now,

$$\begin{aligned} u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} &= u \frac{\partial u}{\partial x} + \cancel{u \frac{\partial v}{\partial y}} \cdot \frac{\partial(uv)}{\partial y} - u \frac{\partial v}{\partial y} \\ &= u \frac{\partial u}{\partial x} + \frac{\partial(uv)}{\partial y} + u \frac{\partial v}{\partial x} \quad (\text{using Eqn 1(iii)}) \\ &= 2u \frac{\partial u}{\partial x} + \frac{\partial(uv)}{\partial y} \\ &= \cancel{2u \frac{\partial u}{\partial x}} + \frac{\partial(uv)}{\partial y} \end{aligned}$$

using this relation Eqn (2) ~~gives us~~ gives us

$$\int_0^{\delta} \left(\frac{\partial u}{\partial t} \right) dy + \int_0^{\delta} \left\{ \cancel{2u \frac{\partial u}{\partial x}} + \frac{\partial(uv)}{\partial y} \right\} dy = -\frac{1}{P} \frac{\partial b}{\partial x} \cdot \delta + \frac{u}{P} \left\{ \left(\frac{\partial u}{\partial y} \right)_S - \left(\frac{\partial u}{\partial y} \right)_0 \right\}$$

$$\int_0^{\delta} \left(\frac{\partial u}{\partial t} \right) dy + \int_0^{\delta} \frac{\partial}{\partial x} (u^2) dy + [uv]_0^{\delta} = -\frac{1}{P} \frac{\partial b}{\partial x} \delta + \frac{u}{P} \left\{ \left(\frac{\partial u}{\partial y} \right)_S - \left(\frac{\partial u}{\partial y} \right)_0 \right\} \quad \text{--- (3)}$$

But if $u = U$ when $y = \delta$

$$[uv]_0^{\delta} = U \cdot v_{\delta} = U \int_0^{\delta} \left(\frac{\partial v}{\partial y} \right) dy = U \int_0^{\delta} \left(- \frac{\partial u}{\partial x} \right) dy$$

also if $\left(\frac{\partial u}{\partial y} \right) = 0$ when $y = \delta$, so Eqn (3) reduce to

$$\int_0^{\delta} \left(\frac{\partial u}{\partial t} \right) dy + \int_0^{\delta} \frac{\partial}{\partial x} (u^2) dy - U \int_0^{\delta} \left(\frac{\partial u}{\partial x} \right) dy = -\frac{1}{P} \frac{\partial b}{\partial x} \delta + \frac{u}{P} \left(\frac{\partial u}{\partial y} \right)_0 \quad \text{--- (4)}$$

now using Leibnitz rule of integration we get

$$\int_0^{\delta} \frac{\partial}{\partial x} \left(\frac{\partial u}{\partial x} \right) dy \leftarrow \frac{d}{dx} \int_0^{\delta} u^2 dy = \int_0^{\delta} \frac{\partial u^2}{\partial x} dy + U^2 \frac{d\delta}{dx} .$$

$$\int_0^{\delta} \frac{\partial u^2}{\partial x} dy = \frac{d}{dx} \int_0^{\delta} u^2 dy - U^2 \frac{d\delta}{dx}$$



$$\text{and } \frac{d}{dx} \int_0^{\delta} u dy = \int_0^{\delta} \frac{\partial u}{\partial x} dy + u \frac{df}{dx}$$

multiply by U both side

$$U \cdot \frac{d}{dx} \int_0^{\delta} u dy = U \int_0^{\delta} \left(\frac{\partial u}{\partial x} \right) dy + U^2 \frac{df}{dx}$$

$$U \int_0^{\delta} \left(\frac{\partial u}{\partial x} \right) dy = U \frac{d}{dx} \int_0^{\delta} u dy - U^2 \frac{df}{dx}$$

$$\text{and } \frac{\partial}{\partial t} \int_0^{\delta} u dy = \int_0^{\delta} \frac{\partial u}{\partial t} dy + \frac{\partial \delta}{\partial t} \cdot U = \int_0^{\delta} \frac{\partial u}{\partial t} dy \\ (\because \delta(x))$$

Therefore Eqn ④ reduces to

$$\frac{\partial}{\partial t} \int_0^{\delta} \left(\frac{\partial u}{\partial x} \right) dy + \frac{d}{dx} \int_0^{\delta} u^2 dy - \cancel{U \frac{df}{dx}} - U \frac{d}{dx} \int_0^{\delta} u dy + U^2 \frac{df}{dx} \\ = -\frac{\delta}{P} \frac{dp}{dx} - \frac{\mu}{P} \left(\frac{\partial u}{\partial y} \right)_{y=0}$$

$$\boxed{\frac{\partial}{\partial t} \int_0^{\delta} u dy + \frac{d}{dx} \int_0^{\delta} u^2 dy - U \frac{d}{dx} \int_0^{\delta} u dy = -\frac{\delta}{P} \frac{dp}{dx} - \frac{1}{P} T_0}$$

Where T_0 is shearing stress at the surface wall.

This is one form of the Von Karman integral relation, it is also known as momentum integral equation of the boundary layer. ~~for steady flow first term is at both side of the boundary layer is zero.~~



Other Forms of the Von Karman Integral Relation :

It is often convenient to have the integral relation in a form in terms of the displacement and momentum thicknesses for steady flow

We know by Euler Eqn of motion outside the boundary layer

$$U \frac{\partial U}{\partial x} = -\frac{1}{\rho} \frac{\partial p}{\partial x}$$

So, Von Karman Integral Eqn becomes

$$\frac{d}{dx} \int_0^{\delta} u^2 dy - U \frac{d}{dx} \int_0^{\delta} u dy - \frac{dU}{dx} \cdot U \delta = -\frac{T_0}{\rho}$$

$$\text{or, } \frac{d}{dx} \int_0^{\delta} u^2 dy - \underbrace{\frac{d}{dx} \left(\int_0^{\delta} U u dy \right) + \left(\frac{dU}{dx} \right) \int_0^{\delta} u dy - \frac{dU}{dx} \int_0^{\delta} U dy}_{\textcircled{1}} = \frac{T_0}{\rho}$$

$$\text{or, } \frac{d}{dx} \int_0^{\delta} u (U-u) dy + \frac{dU}{dx} \int_0^{\delta} (U-u) dy = \frac{T_0}{\rho}$$

taking limit $\delta \rightarrow \infty$ we get

$$\frac{d}{dx} \int_0^{\infty} u (U-u) dy + \frac{dU}{dx} \int_0^{\infty} (U-u) dy = \frac{T_0}{\rho}$$

$$\boxed{\frac{d}{dx} (U \delta_2) + \frac{dU}{dx} \cdot U \delta_1 = \frac{T_0}{\rho}} \Rightarrow \boxed{\frac{d\delta_2}{dx} + (2\delta_2 + \delta_1) \frac{1}{U} \frac{dU}{dx} = \frac{T_0}{\rho U^2}}$$

where δ_1 & δ_2 are ~~momentum~~ displacement and momentum thickness. Given by $\delta_1 = \int_0^{\infty} \left(1 - \frac{u}{U}\right) dy$

$$\text{& } \delta_2 = \int_0^{\infty} \frac{u}{U} \left(1 - \frac{u}{U}\right) dy$$



Example:

For flow over a flat plate, the velocity profile within the boundary layer is assumed as $u = U \tanh(\frac{y}{\delta})$. Show that the skin friction coefficient on the flat plate is

given by :

$$c_f = \frac{T_0}{\rho U^2} = \left(\frac{0.392}{0.3125} \right) \left(\frac{U x}{v} \right)^{-\frac{1}{2}}$$

where symbols have their usual meanings.

Sol: The von Karman integral relation is

$$\frac{d}{dx} \int_0^\delta u(U-u) dy + \frac{du}{dx} \cdot \int_0^\delta (U-u) dy = \frac{T_0}{\rho}$$

here $U = \text{const.}$ so above eqn reduces to

$$\frac{d}{dx} \int_0^\delta u(U-u) dy = \frac{T_0}{\rho}$$

$$\frac{d}{dx} \int_0^\delta \frac{U}{U} \left(1 - \frac{u}{U}\right) dy = \frac{T_0}{\rho U^2}$$

$$\frac{d}{dx} \frac{T_0}{\rho U^2} = \frac{d}{dx} \int_0^\delta \frac{U}{U} \left(1 - \frac{u}{U}\right) dy$$

$$\frac{T_0}{\rho U^2} = \frac{d}{dx} \int_0^\delta \tanh\left(\frac{y}{\delta}\right) \left\{ 1 - \tanh\left(\frac{y}{\delta}\right) \right\} dy$$

$$\frac{T_0}{\rho U^2} = \frac{d}{dx} \int_0^1 \tanh\eta \cdot (1 - \tanh\eta) s d\eta \quad (\text{let } \eta = \frac{y}{\delta})$$

$$= \frac{d\delta}{dx} \cdot \int_0^1 [\tanh\eta - (1 - \tanh^2\eta)] d\eta$$

$$= \frac{d\delta}{dx} \cdot [\log \tanh\eta - \eta + \tanh\eta]_0^1$$



$$\frac{\tau_0}{\rho v^2} = \frac{d\delta}{dx} \cdot \left[\log \cosh \gamma - \log \gamma^b - \gamma + \tanh \gamma \right]$$

$$= \frac{d\delta}{dx} \cdot [0.433781 - 1 + 0.761594]$$

$$\frac{\tau_0}{\rho v^2} = \frac{d\delta}{dx} \cdot (0.1953752) \quad \text{--- (1)}$$

now since

$$\begin{aligned}\tau_0 &= \mu \left(\frac{\partial u}{\partial y} \right)_{y=0} \\ &= \mu \left[\frac{\partial}{\partial y} v \tanh \left(\frac{y}{\delta} \right) \right]_{y=0} \\ &= \mu v \left[\frac{1}{\delta} \operatorname{sech}^2 \left(\frac{y}{\delta} \right) \right]_{y=0}\end{aligned}$$

$$\tau_0 = \mu v \cdot \frac{1}{\delta} \quad \text{--- (2)}$$

By (1) & (2) we have

$$\frac{(\mu v)}{\rho v^2} = \frac{d\delta}{dx} \cdot (0.1953752)$$

$$\frac{\mu}{\rho \delta v} = \frac{d\delta}{dx} \cdot (0.1953752)$$

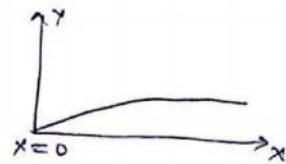
$$\frac{v}{U} = \delta \frac{d\delta}{dx} \cdot (0.1953752)$$

$$\delta \frac{d\delta}{dx} = \frac{v}{U} \cdot \frac{dx}{(0.1953752)}$$



$$\frac{f^2}{2} = \frac{\nu}{U} (5.118358) x + C$$

When $x=0$, $\delta=0$ i.e. $C=0$



$$\text{So, } \delta = \sqrt{2 \times 5.118358} \cdot \sqrt{\frac{\nu x}{U}}$$

$$\delta = 3.199487 \cdot \sqrt{\frac{\nu x}{U}}$$

$$\frac{d\delta}{dx} = \frac{3.199487}{2} \sqrt{\frac{\nu}{U} x}$$

Substituting this value in Eqn(i)

$$\frac{T_0}{\rho U^2} = \frac{(0.1953752), (3.199487)}{2} \left(\frac{Ux}{\nu}\right)^{\frac{1}{2}}$$

$$C_f = \frac{T_0}{\rho U^2} = 0.31255 \left(\frac{Ux}{\nu}\right)^{\frac{1}{2}}$$

↓ skin friction coefficient

For velocity profiles are

For the following velocity profiles in the boundary layer

$$(i) u = U \left[2 \frac{y}{\delta} - \left(\frac{y}{\delta} \right)^2 \right]$$

$$(ii) u = U \sin \left\{ \frac{\pi}{2} \left(\frac{y}{\delta} \right) \right\}$$

Obtain the expression for δ_1 , δ_2 and T_0 .

